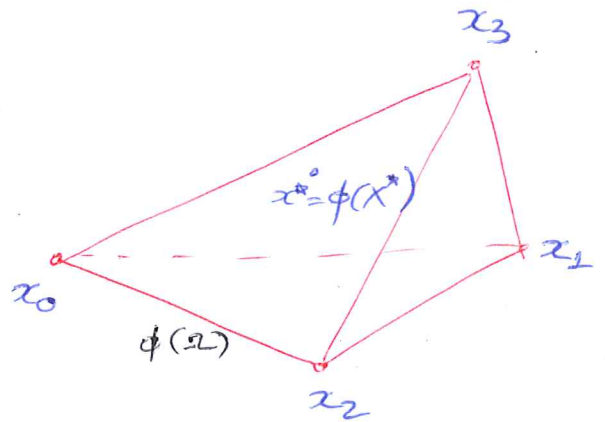
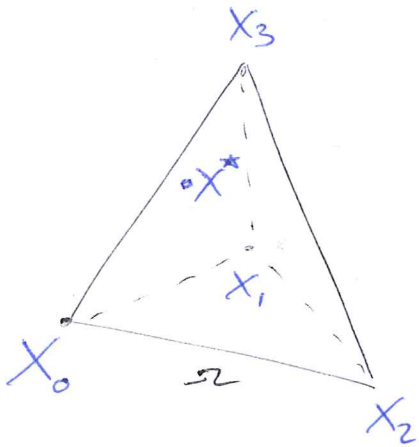
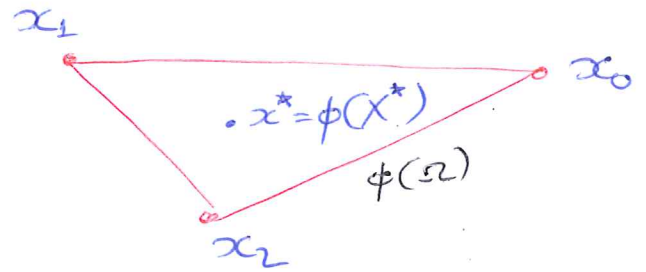
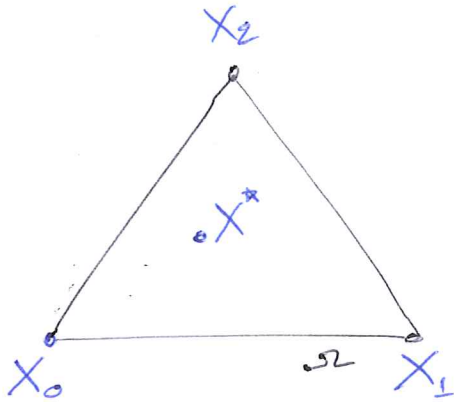


As a building block of volumetric objects, we consider a triangle (2D) or a tetrahedron (3D)



The energy stored (as deformation-induced potential energy) on each triangle/tetrahedron is expressed as a volume integral

$$E = \int_{\Omega} \psi(F(x)) dx$$

where $\psi()$ is an energy density function and

$F = \frac{\partial \phi}{\partial X}$ is the Jacobian matrix of $\phi(C)$, and called the deformation gradient

→ This definition automatically makes the energy be translation-invariant.

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→ If, additionally, $\Psi(F) = \Psi(RF)$ for all F and any rotation matrix R , the material is rotation-invariant.

→ Isotropic models satisfy $\Psi(F) = \Psi(FR)$ as well.

Practical computation

Since we don't know $\phi(X)$ analytically, we approximate it as follows:

→ Assume that $\phi(X) = FX + t$ within each triangle/tetrahedron
(F is the deformation gradient).

→ By setting $x_0 = \phi(X_0), \dots, x_3 = \phi(X_3)$ we arrive at the numerical expression

$$F = D_S D_m^{-1}$$

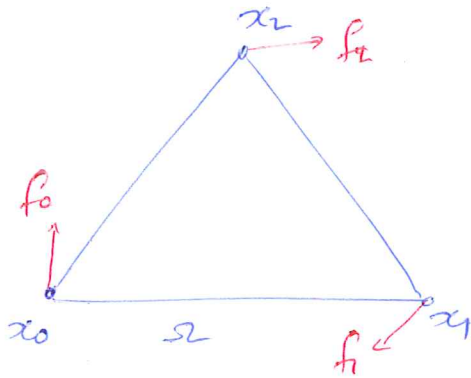
Where D_S : the deformed shape matrix (changes over time)

$$D_S = \begin{bmatrix} \vec{x}_1 - \vec{x}_0 & | & \vec{x}_2 - \vec{x}_0 \end{bmatrix} \in \mathbb{R}^{2 \times 2} \quad (2D) \quad D_S = \begin{bmatrix} \vec{x}_1 - \vec{x}_0 & | & \vec{x}_2 - \vec{x}_0 & | & \vec{x}_3 - \vec{x}_0 \end{bmatrix} \in \mathbb{R}^{3 \times 3} \quad (3D)$$

D_m : the reference shape matrix (never changes, can be precomputed)

$$D_m = \begin{bmatrix} \vec{X}_1 - \vec{X}_0 & | & \vec{X}_2 - \vec{X}_0 \end{bmatrix} \in \mathbb{R}^{2 \times 2} \quad (2D) \quad D_m = \begin{bmatrix} \vec{X}_1 - \vec{X}_0 & | & \vec{X}_2 - \vec{X}_0 & | & \vec{X}_3 - \vec{X}_0 \end{bmatrix} \in \mathbb{R}^{3 \times 3} \quad (3D)$$

Force computation



In theory

* Calculate element energy

$$E(\Omega) = \int_{\Omega} \psi(F) dX =$$

\hookrightarrow const on Ω

$$= \text{Vol}(\Omega) \cdot \psi(F^{\Omega})$$

$$= E^{\Omega}(x_0, x_1, x_2)$$

* Forces are the negative gradients of the energy

$$\vec{f}_i = -\partial E / \partial \vec{x}_i$$

In practice

The derivation above (after lots of calcul) reduces to the following compact form:

Define Force matrix G :

$$G = \begin{bmatrix} \vec{f}_1 \\ \vec{f}_2 \end{bmatrix} \text{ in 2D} \quad G = \begin{bmatrix} \vec{f}_1 & \vec{f}_2 & \vec{f}_3 \end{bmatrix} \text{ in 3D}$$

(Note: \vec{f}_0 is excluded from G , in both cases)

Then, the following expression holds:

$$G = -\text{Vol}(\Omega) P(F) D_m^{-T}$$

Once f_1, f_2, f_3 (in 3D) have been computed,

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f_0 can be computed by noting that $\sum f_i = 0$, i.e.:

$$\bullet \vec{f}_0 = -\vec{f}_1 - \vec{f}_2 \quad (2D)$$

$$\bullet \vec{f}_0 = -\vec{f}_1 - \vec{f}_2 - \vec{f}_3 \quad (3D)$$

Notes:

- PCF) is a matrix called the 1st Piola-Kirchhoff stress tensor. Given a formula for $\Psi(CF)$, the expression for PCF) is uniquely determined
- The volume $\text{Vol}(\Omega)$ of the undeformed triangle/tetrahedron is given by the formula

$$\text{Vol}(\Omega) = \begin{cases} \frac{1}{2} |\det D_m| & \text{in 2D} \\ \frac{1}{6} |\det D_m| & \text{in 3D.} \end{cases}$$

- Many components of the formula $G = -\text{Vol}(\Omega) \text{PCF}) D_m^{-T}$ are constants and can be precomputed. In fact we can write $G = P \cdot B_m$, where

$$B_m = \begin{cases} -\frac{1}{2} |\det D_m| D_m^{-T} & \text{in 2D} \\ -\frac{1}{6} |\det D_m| D_m^{-T} & \text{in 3D.} \end{cases}$$

Common material definitions

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For all cases below:

μ & λ (Lamé parameters) are functions of the Young's modulus and Poisson's ratio

Linear Elasticity

$$\varepsilon = \frac{1}{2}(F + F^T) - I \quad \text{"Small strain tensor"}$$

(Compare with $(\frac{l}{l_0} - 1)$ for springs)

$$\Psi = \mu \|\varepsilon\|_F^2 + \frac{\lambda}{2} \text{tr}(\varepsilon)^2$$

(Note the definition: $\|M\|_F^2 = \sum_{ij} M_{ij}^2$ "Frobenius norm")

$$P(F) = 2\mu \varepsilon + \frac{\lambda}{2} \text{tr}(\varepsilon) \cdot I$$

Properties

* $F(x_0, x_1, x_2, x_3)$ is linear function of vertex positions $\underline{x} = (x_0, \dots, x_3)$

$\Rightarrow \varepsilon$ is affine on $\underline{x} \Rightarrow P$ is affine on $\underline{x} =$

$\Rightarrow G$ is affine on $\underline{x} \Rightarrow$ Forces $\underline{f} := K\underline{x} + b$

This makes computation cheap (K can be precomputed, is a constant matrix) and facilitates Backward Euler

* Linear elasticity is not rotationally invariant

\Rightarrow Significant visible artifacts for large deformations.

St. Venant Kirchhoff material

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→ Replaces small strain tensor with :

$$E = \frac{1}{2} (F^T F - I) \quad \text{The "Green strain tensor"}$$

$$\Psi(F) = \mu \|E\|_F^2 + \lambda \text{tr}^2(E)$$

$$P(F) = F \left\{ 2\mu E + \lambda \text{tr}(E) \cdot I \right\}.$$

Properties

⇒ Isotropic and rotation-invariant.

⇒ $f(\underline{x})$ is a nonlinear (in fact, a cubic polynomial) function (Backward Euler is tricky, computation is more expensive).

⇒ Issues : This material has several "nonphysical" rest states, e.g.

• When $F = -I$, or in general $F = R \begin{pmatrix} \pm 1 & & \\ & \pm 1 & \\ & & \pm 1 \end{pmatrix}$
(any "flipped" configuration), we have

$$E = 0 \rightsquigarrow P = 0 \rightsquigarrow f = 0 \quad (\text{No forces!})$$

• When $F = 0$ (everything compressed to a point)

we also have $P = 0 \Rightarrow$ No force to trigger restoration.

⇒ Good for rotational invariance, but not for dramatic shape change!

Co-rotational Linear Elasticity

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→ Tries to mimic linear elasticity as much as possible while "fixing" rotational invariance issue.

→ Based on "Polar decomposition":

Def Any matrix $A \in \mathbb{R}^{n \times n}$ can be decomposed as

$$A = R \cdot S \leftarrow \text{Symmetric.}$$

↑ Rotation matrix, $R^T R = I$

Practical Implementation: If we know the SVD of A

$$\begin{aligned} A &= U \Sigma V^T \\ &= \underbrace{U V^T}_{=: R} \underbrace{V \Sigma V^T}_{=: S} = RS. \end{aligned}$$

Corotated formulation:

→ Let $F = RS$ be the polar decomposition of F

→ $\epsilon^R = S - I$ "Rotated strain tensor"

$$\rightarrow \psi(F) = \mu \|\epsilon^R\|_F^2 + \frac{\lambda}{2} \text{tr}(\epsilon^R)^2$$

$$\rightarrow P(F) = R \left\{ 2\mu \epsilon^R + \lambda \text{tr}(\epsilon^R) I \right\}$$

Properties

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- ⇒ Isotropic and rotation invariant!
- ⇒ Nonlinear! (R & S are nonlinear functions of x !)
- ⇒ Approximations (linear) that facilitate Backward Euler do exist ⇒ "Warped stiffness" techniques.
- ⇒ Bulk of cost = Computing the SVD or polar decomposition of F
- ⇒ Does not resist compression too strongly: A finite force would suffice to collapse a body to a single point.
(i.e. $F=0 \Rightarrow f = \text{finite}$)

In the real world as $F \rightarrow 0$, we would have $\psi \rightarrow \infty$
 $f \rightarrow \infty$.

Neo-hookean Material (or, Neo-hookean elasticity)

$$\Psi(F) = \frac{\mu}{2} [\text{tr}(F^T F) - \text{tr}(I)] - \mu \log(J) + \frac{\lambda}{2} \log^2(J)$$

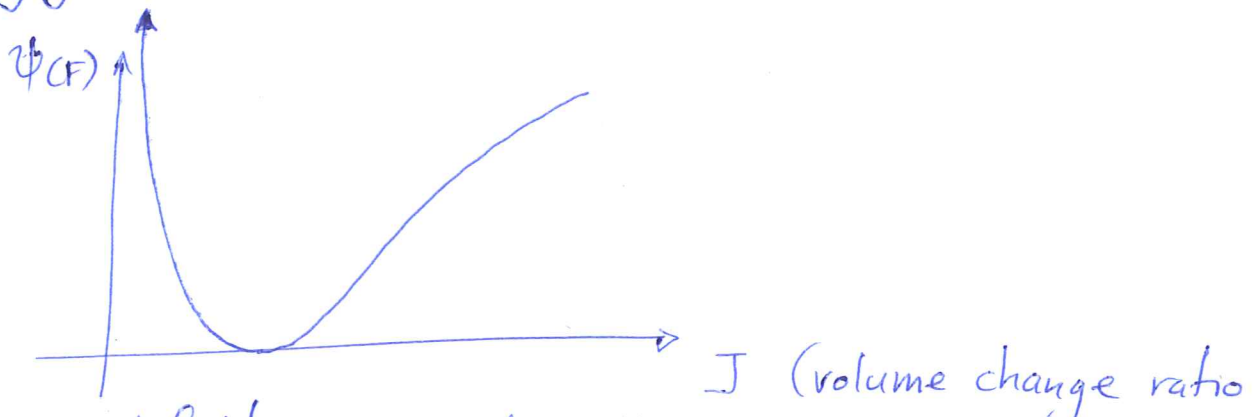
$$P(F) = \mu (F - F^{-T}) + \lambda \log(J) F^{-T}$$

Here $J = \det(F)$ = volume change ratio = $\frac{\text{Volume of } \phi(\Omega)}{\text{Volume of } \Omega}$

(Incompressibility implies $J=1$)

Properties

- ⇒ Nonlinear, isotropic, rotation-invariant
- ⇒ Very good approximation of natural rubber-like materials (and reasonably good approximation of flesh).
- ⇒ Strongly resists compression to zero volume



(It requires infinite energy to collapse a material to a single point)

- ⇒ The last property, although faithful to reality, creates issues with robustness:

If $J \ll 1$ forces are extremely strong

(may need to use CFL number $\ll 1$ for stability)

If $J < 0$ forces are undefined! ⇒ Simulation Halts!

⇒ Could arise from very fast moving boundary trajectories

⇒ Could arise from nonphysical artist-imposed kinematic constraints.

Consequence:

- When using neo-hookean we may be forced to move kinematic constraints slowly or may be limited to simulations without nonphysical constraints
- Remedy: Invertible methods (see ref.), or just use co-rotated elasticity.

Damping

Commonly used: Linear damping model

Define: $\dot{D}_S = \begin{bmatrix} \vec{v}_1 - \vec{v}_0 & | & \vec{v}_2 - \vec{v}_0 & | & \vec{v}_3 - \vec{v}_0 \end{bmatrix}$

$$\dot{F} = \dot{D}_S \cdot D_M^{-1}$$

$$\epsilon^d = \frac{1}{2}(\dot{F} + \dot{F}^T)$$

$$P^d = 2\alpha \epsilon^d + \beta \text{tr}(\epsilon^d) \cdot I$$

$$G^d = -\text{Vol}(\Omega) P^d D_M^{-T} = \begin{bmatrix} f_1^d & | & f_2^d & | & f_3^d \end{bmatrix}$$

As before: $f_0^d = -f_1^d - f_2^d - f_3^d$

Parameter Selection

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We typically set:

$$\alpha = \gamma \cdot \mu \quad \beta = \gamma \cdot \lambda$$

$\gamma :=$ Rayleigh coefficient.

Stability We treat γ as we would for the ratio b/k on a single spring.

i.e. if $\Delta t < 2k/b$ for a single spring

then $\Delta t < 2/\gamma$ for the volumetric forces